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# Computer programs for ultimate strength of longitudinally stiffened panels (large $b/t$ ), August 1967

J. F. Vojta

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LS Beedle

Built-Up Members in Plastic Design

COMPUTER PROGRAM FOR ULTIMATE STRENGTH OF  
LONGITUDINALLY STIFFENED PANELS

(LARGE  $b/t$ )

by

Joseph F. Vojta

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Fritz Engineering Laboratory  
Department of Civil Engineering  
Lehigh University  
Bethlehem, Pennsylvania

August, 1967

Fritz Engineering Laboratory Report No. 248.20

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### INTRODUCTION

Longitudinally stiffened plate panels are commonly used as structural elements of ship hulls (Fig. 1). A knowledge of the structural behavior of the plate panels is thus very important in ship design. For this reason research studies have been conducted at Lehigh University to determine the ultimate strength of longitudinally stiffened plate panels. (1)(2)(3)(4) Computer programs have been used to perform the many numerical operations involved in using the theories developed by the research studies. (1)(3)(5)

This report describes a computer program that analyzes a stiffened plate panel and determines the maximum fixed and simply-supported lengths that it can have under the given loading conditions. The cross section considered is shown in Fig. 2. The computer program in this report analyzes panels where the main element of the cross section is a plate having the ratio of stiffener spacing to plate thickness ( $b/t$ ) sufficiently large ( $\geq 45$ ) so that the plate buckles before the ultimate load is reached. The program in this report was used extensively in obtaining the data for the ultimate strength design curves of References 3 and 4.

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\*The loading conditions are given in more detail in Ref. 3.

\*\*More information concerning the cross section, residual stress condition, and plate buckling action is contained in Ref. 3.

\*\*\*Reference 5 contains a computer program which analyzes, with more accuracy, the case where  $b/t$  is less than 45.

The analysis performed by the computer program is essentially in two parts. The main program evaluates the response of the cross section to the given loading conditions in the form of a moment-curvature-thrust,  $(M-\theta-P)$ , curve for a given axial load. This curve is then used in a numerical integration of small segments which yields the maximum fixed and simply-supported lengths for each of a series of mid-span starting curvatures. From the plot of maximum lengths vs. the mid-span starting curvatures, Fig. 3, the program uses a parabolic interpolation technique to obtain the maximum length the panel can have under the given loading condition. Separate curves and maximum lengths are found for the fixed and simply-supported ends.

The analysis considers non-linear effects, non-symmetrical cross sections, and inelastic behavior. The effects of residual stresses in the plate and different yield stresses in the plate and stiffener are also considered.

Some important factors concerning the use of the computer program are:

- 1) The program continuously loops the input parameters as specified by the user. Thus a series of cross sections can be obtained with a minimum amount of input cards required.
- 2) The program is self regulatory in that important control values can be input as constant values for all cross sections to be run. The program internally modifies these values to adjust for changes in the cross section for the particular case being analysed.

- 3) The input and the resulting output are printed and are also punched on computer cards. The punched results facilitate further use and arrangement of the data.
- 4) Provision has been made to label each run with an identifying number chosen by the user.
- 5) When the input is such that the no meaningful results can be obtained the program proceeds to the next case and prints out the reason why the particular case could not be run.\*

This report deals with the preparation of input data for the program, technical information about the program and its use, and an explanation of the printed and punched output. The appendix presents the program and an example of the output. In arranging this report an attempt has been made to conform to the standards of Ship Design Division Instruction 10462 of the Bureau of Ships, U. S. Navy.

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\*One particular case of importance occurs when the axial load is very high for certain cross sections. The moment-curvature curve in this case has only negative (compression in plate due to bending) bending moments for all values of curvature. This means that under only axial load the section requires an application of an external moment in order to obtain equilibrium. The computer program cannot handle this case and this information is printed in the results.

## PART I - IDENTIFICATION

1. Title: Computer Program for Ultimate Strength of Longitudinally Stiffened Panels (Large  $b/t$ )
2. Brief Description: An  $m-\phi-P$  curve is computed for the cross section being analyzed. Then, for the given loading the program makes successive computations of both the simply-supported and fixed end panel lengths for each of a series of mid-span curvatures. By comparing each new set of lengths with the previous set, the maximum set is determined.

The program consists of four main parts:

- 1) The MAIN program which computes the  $m-\phi-P$  curve for constant axial load for the cross section being analyzed.
- 2) Subroutine INTEG which numerically integrates a series of small segments to determine the simply-supported and fixed end lengths for a given combination of axial and lateral load. This is done for a series of mid-span curvatures and the maximum length values are interpolated from the curve that results.
- 3) Function BC which finds the zero root of a parabolic equation by using Newton's Method for Finding Zeros.
- 4) Function VAL which calculates an intermediate point on a parabola when given three points.
- 5) Subroutine PARAM which regulates the increment in curvature required before printing a point on the  $m-\phi-P$  curve. This subroutine also adjusts the increment of plate strain depending on the cross section being used.

- 6) Function CUR which provides an estimate of the initial mid-span curvature.
- 7) Subroutine VALUES which regulates the increments of center curvature and of segment length to adjust for the case being run.

Input data are read directly from cards into the main program. The main program will iterate through successive sets of data and will be used for each cross section specified on the cards. After each time through the main program, subroutine INTEG will be called and will be cycled for each lateral load specified in the input for the given cross section. The other subprograms are called for and used at various times throughout the use of the main program and subroutine INTEG.

3. a) Author: Jun Kondo,\* Tsuneo Tsuiji,\* Joseph F. Vojta, and Alexis Ostapenko  
 b) Date: August, 1967
4. Code: Fortran II
5. Machine: CR 225 (any other machine that compiles Fortran II may be used)
6. Security Classification: Unclassified
7. Estimated Running Time: Main program 90 sec.  
 INTEG\*\* 50 sec.  
 140 sec.

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\*Developed part of this program before it was modified into its present form.

\*\*This time is estimated for each lateral load



## PART II - PURPOSE & METHOD

1. Description of Theory: See ULTIMATE STRENGTH DESIGN OF LONGITUDINALLY STIFFENED PLATE PANELS WITH LARGE  $b/t$ , by Joseph F. Vojta and Alois Ostapenko, Fritz Engineering Laboratory Report No. 248.18, Lehigh University, 1965.
2. Assumptions:
  - a) No local buckling of any elements except the main plate element.
  - b) The plate is assumed to be simply-supported at the stiffeners.
  - c) The distribution of the residual stresses in the plate is assumed to be rectangular. The residual stresses in the flange have a small effect and are neglected.
  - d) The postbuckling action in the plate is assumed to be governed by Koiter's equation until its maximum stress is reached.
3. References: See REFERENCES at the end of this report.

## PART III - RESTRICTIONS

1. General Restrictions: None
2. Limitations for Use:
  - a) The program will not run for extremely high axial loads where the bending moment is negative for all values of curvature. The program exits and prints "X: Positive Bending Moment Capacity (CMX1 is negative) CMX1 = XXX.XXXX".

---

\*For further information see Ref. 3.

b) Acceptable results are not produced in all cases when the mid-span curvature for the simply-supported length is less than the mid-span curvature for the corresponding fixed-end length. This is denoted when the computer prints "ERROR\*\*IS EXCEEDS 2". Under normal conditions this will not occur or will occur very infrequently.

3. Nonstandard Hardware & Tapes: None

4. Maximum Array Sizes: 7 Arrays are used: FI (200)

GI (200)

P (30)

R (20)

Q (10)

Y (30)

X (25)

#### PART IV - NONSTANDARD MACHINE OPERATING INSTRUCTIONS

1. Special Operating Instructions: None

2. Restart Instructions: None

3. Error Correction: None

#### PART V - DATA PREPARATION

1. Card Input Form:

<u>Card</u>	<u>Format</u>	<u>Variable Range</u>	<u>Comment</u>
1	13	NN	Number of data sets to be run (i.e. times computer must read through cards 2 - 13)

<u>Card</u>	<u>Format</u>	<u>Variable Names</u>	<u>Comment</u>
2	I5, I2	NRUN, IALL	Case number, number of first the lateral load, respectively.
3	F7,0	R(6)	Modulus of elasticity (ksi)
4	"	EX	Nondimensional strain increment, $\frac{\Delta \epsilon}{\epsilon_{cr}}$ (Use about 0.003 for most cases)
5	2F7,6	DSI, FIG	Nondimensional segment length and nondimensional increment of mid-span curvature. $\frac{4.9}{r} \rho_{yp}$ , $\left(\frac{\theta}{\epsilon_{cr}}\right)_0$ (Generally use 0.12, 0.20)
6	2F7.2,3I2, 2F7.2, I2	QL1,QLIC,NSUM, LT,IALLA,QLIA, QLICA,NSUMA	First three values for "loop" of QLL, then punch 1 for another "loop", 0 for no other loop, the lateral load number for the second "loop", then last three values for the second "loop". $QLL = q \left(\frac{d}{t}\right)$ where q is in psi.
7	2F7.2, I2	R4N,R4I,NR4	Stiffener yield stress (ksi) ("loop" of R(4))
8	" "	R5N,R5I,NR5	Plate yield stress (ksi)("loop" of R(5))
9	" "	ASTN,ASTI,NAST	Nondimensional area of stiffener ("loop" of AST)
10	" "	AFSN,AFSI,NAFS	Nondimensional area of flange ("loop" of AFS)
11	" "	RESN,RESI,NRES	"Loop" of compressive residual stress $\frac{\sigma}{\sigma_{yp}}$
12	" "	BN,BI,NBN	"Loop" of b/t
13	" "	R1N,R1I,NR1	"Loop" of axial load ( $P/P_{cr}$ )

\*"loop" means first value, increment of value, total number of values to be used (including the first value). For example 2.12 0.50 4 means run the values 2.12, 2.62, 3.12 and 3.62.

If card 1 has a value greater than 1, cards 7 to 13 must be repeated the required number of times.

Last card - Any illegal character on a data card will terminate the run. Such an illegal character can be END OF DATA.

## 2. Sample Input:

### Card No.

1	1
2	77 1
3	29600
4	* 0.003
5	0.1200 0.20000
6	** 40.0 140.0 3 1 6 110.0 20.0 1
7	** 47.0 33.0 1
8	** 47.0 33.0 1
9	*,** 0.34 0.14 2
10	*,** 0.60 0.05 1
11	** 0.075 0.075 1
12	** 110.0 10.0 1
13	** 1.3 0.1 1

\*This does not conform to the specified format. However, the decimals punched in the input data cards have priority over the decimal location specified in the input format.

\*\*The input value specifying the increment to be used has no meaning because the last input (1) specifies no "loops". Any arbitrary value can thus be used for the increment.

The example run shown in the appendix uses this data. However card 9 specifies that the run is to continue on for a new value of AST. Thus run number 78 would be similar to run 77 except that it would have a value of 0.48 for AST.

Card 6 describes the looping of the lateral load parameter QLL. The values of QLL will be 40, 180, and 320 for the first loop corresponding to lateral load numbers 1, 2, and 3, (see card 2). Then it loops again (1 in column 18) and gives a value of QLL = 110 for lateral load number 6 (column 20).

### 3. Output Form Description:

<u>Page</u>	<u>Comment</u>
1	<p>Lists input and some computed values for checking and later identification. Some of the values that are not so easily identified are:</p> <p>AF = area ratio, flange to total</p> <p>AW = area ratio, web to total</p> <p>AP = area ratio, plate to total</p> <p>GT = nondimensional distance from the plate to the centroidal axis</p> <p>SM = nondimensional section modulus, <math>\frac{S}{Ad}</math></p> <p>MAX LOAD = maximum external panel load</p> <p>STIFF YIELD = ratio of stiffener yield stress to critical stress</p> <p>PLATE YIELD = ratio of plate yield stress to critical stress</p> <p>MAX PL STRESS = nondimensional plate stress computed from Koiter's Equation<sup>(9)</sup></p>

PageComment

In addition the printouts on page 1 include points on the moment-curvature plot for the various corresponding values of plate strain,  $\frac{\epsilon}{\epsilon_{cr}}$  \*.

2,3,44

Continues listing of 200 points the moment-curvature plot.

5

Lists results for each parameter QLL. Values for both the maximum ultimate lengths for both the simply-supported and fixed end cases are printed out. Also listed are the corresponding mid-span curvature, end moment, and axial load at the ends, all nondimensionalized by the critical value. In addition, listings are made of the total panel length, chord length, and center deflection all nondimensionalized by the radius of gyration of the idealized cross section\*\*. The slope in radians is printed.

In addition to the errors listed in Part XII, the following error messages are printed if the error does occur.

MessageComment

a) LL EXCEEDS 10

This is a check on the internal computations in the m- $\phi$ -P calculations. If this occurs the results are not correct.

\*The symbols used, if not defined, correspond to those used in Reference 3.

\*\*See Ref. 3 for "idealized" cross section.

- b) **INITIAL PFC TOO HIGH** The initial estimate of mid-span curvature is too high. Revisions in Function CUR are then required.
- c) **TOO MANY PFC VALUES REQUIRED. USE A LARGER FIC** The increment in mid-span curvature is too small. Increase the value of FIC in input card 5.
- d) **TOO MANY LENGTH INCREMENTS** Increase DSI in input card 5. The increment of segment length is greatly reduced for segments computed after the simply-supported length has been computed but the fixed end length is being calculated for a particular mid-span curvature. This type of error becomes a problem for very high axial loads because then the simply-support length is low but the fixed end length is high and many segment lengths are required.
- e) **CURVATURE OUTSIDE LIMITS OF M-PHI-P CURVE** The limits of the maximum curvature values computed in the M- $\phi$ -P plot are not wide enough. To remedy this, increase the value of RX in input card 4.

#### 4. Symbol List and Definitions

AST	Area of Stiffener divided by area of plate
ASTI	Increment of AST
ASTH	Initial value in "loop" of AST

AFS	Area of flange divided by area of stiffener
AFSI	Increment of AFS
AFSN	Initial value in "loop" of AFS
B	Width-thickness ratio (b/t) of plate
BC1, BC2, BC3, BC4	Dummy variables in Function BG
BI	Increment of B
BN	Initial value in "loop" of B
CA	Dummy variable in Function VAL, intermediate value in Subroutine INTEG
CS	In the main program CS is the width of the tensile residual stress zone divided by the flange width; dummy variable in Function VAL; intermediate value in Subroutine INTEG
CC	Mid-span curvature corresponding to the ultimate condition, i.e. $(\phi/\phi_{cr})_{ULT}$ where $\phi_{cr} = \frac{\epsilon_{cr}}{\alpha d}$ , and $\alpha$ = nondimensional distance from plate to centroidal axis
CD	Computed axial load at the end of the panel divided by the critical axial load $P_{cr}$ , where $P_{cr} = (\text{panelarea, } A) \times (\text{critical buckling stress, } \sigma_{cr})$
CM	Moment array, Bending moment divided by the critical moment for the main program and by the yield moment for subroutine INTEG
CMX1, CMX2	Positive and negative bending moment capacities of the cross section (nondimensional)



GR	Nondimensional radius of gyration
GRS	Plate compressive residual stress divided by the critical buckling stress, $\sigma_{cr}$
CS	Cosine of slope angle
G11, G12	Dummy variables in Function VAL
DEFA	Difference in curvatures
DSI	Increment of segment length (see Part V)
DSIA	Modifying factor for increment of segment length
DSII	Modified increment of segment length (Adjust for specific cross section being analyzed)
DSI	Intermediate segment length value
EFIX	Final maximum panel length for fixed end case, $(\frac{L}{r})^2_{max}$
EMOM	End moment corresponding to the maximum ultimate condition. Also used as a storage location for various values of differences in sines of slope angles
EPIN	Final maximum panel length for simply-supported ends, $(\frac{L}{r})^2_{max}$
EY	Plate yield strain, $\frac{\epsilon_{yp}}{\epsilon_{cr}}$
FI	Curvature array. Curvature divided by critical curvature in the main program, and by the yield curvature in Subroutine INTEG
FIG	Increment of mid-span curvature (see Part V)
FICA	Modifying factor for increment of mid-span curvature
FII	Modified increment of mid-span curvature (modified for the particular cross section being analyzed and non-dimensionalized by the critical curvature)

FOI	Modified increment of mid-span curvature (modified for the particular cross section being analyzed and nondimensionalized by the critical curvature)
I	Counter of how many terminal curvatures have been assumed
IA	Counter in main program and in Subroutine INTEG
IATLL	Storage location for number assigned to the original IALL Value
IALL	Number assigned to the first lateral load in the first "loop" of lateral loads
IALLA	Number assigned to the first lateral load in the second "loop" of lateral loads
IB	Counter to determine which end conditions have been determined (S.S. or fixed)
IC	Counter to see if the ultimate simply-supported length has already been determined
ID	Similar to IC but for each assumed mid-span curvature
IE	Counter for determining if the simply-supported length has already been determined for each assumed mid-span curvature
II	Counter of lateral loads used in each "loop" of lateral loads
INUM	Number of the lateral load presently being run
J	Counter that relates the curvature equation to its corresponding moment equation
JA	Counter of how many mid-span curvature increments have been used
JB	Counter of how many segment length increments have been used

JAPS	Counter of how many values of AFS have been used
JAST	Counter of how many values of AST have been used
JBB	Counter of how many values of B have been used
JRES	Counter of how many values of RES have been used
JR1	Counter of how many values of R(1),(i.e. axial load), have been used
JR4	Counter of how many values of R(4),(i.e. stiffener yield stress), have been used
JR5	Counter of how many values of R(5),(i.e. plate yield stress), have been used
K	General counter
KK	Counter of how many trial slopes for Newton's Method have been used in Function BC
L	Counter of how many times curvatures are computed and compared to assumed curvatures
LL	Counter to check for errors
LEUN	Number of the case being run (n-P-P)
LS	Counter of how many complete lateral load "loops" were run
LT	Amount of excess lateral load "loops" prescribed in input
N	Counter of points on n-P-P curve. 100 points for negative bending and then 100 points for positive bending
NAPS	Number of values in "loop" of AFS
NAST	Number of values in "loop" of AST
NBB	Number of values in "loop" of B
NBGM	Number of values in original "loop" of QLL

NN	Number of sets of input data cards
NRES	Storage location for the number of values in "loop" of RES
NRIUN	Number assigned to original run number
NR1	Number of values in "loop" of R(1)
NR4	Number of values in "loop" of R(4)
NRS	Number of values in "loop" of R(5)
NSRM	Number of values in "loop" of QLL (original loop)
NSRMA	Number of values in "loop" of QLL (second loop)
P	Array which generally is used to store stresses and strains
PHC	Mid-span curvature, $\theta/\theta_{cr}$
PHC1	Mid-span curvature, $\theta/\theta_{yp}$
PNT1	Nondimensional distance ( $\epsilon$ ) from the plate to the neutral axis
PSX1	Maximum plate load corresponding to the maximum positive moment
PSX2	Maximum plate load corresponding to the maximum negative moment
Q	Array of relative area ratios
QFLIC	Storage location for lateral load increment in the first lateral load loop
QFL1	Storage location for first lateral load value in first lateral load "loop"
QIR	$\frac{gbE_{yd}}{2A}$ $\sigma_{yp}$

where

$q$  = hydrostatic lateral pressure (psi)

$b$  = plate width or stiffener spacing

$E$  = modulus of elasticity

$\alpha$  = nondimensional distance from the plate to the  
centroidal axis

$d$  = depth of stiffener (in.)

$\sigma_{yp}$  = yield stress in the stiffener

$A$  = area of portion of panel being analyzed

QLIC Increment of lateral load parameter QLL for first "loop"  
QLICA Increment of lateral load parameter QLL for second "loop"  
QLL Lateral load parameter  $q$  ( $d/t$ )

where

$q$  = see QIR

$d/t$  = ratio of depth of stiffeners to thickness of  
plate

QL1 First lateral load parameter in first "loop" of lateral  
load

QL1A First lateral load parameter in second "loop" of  
lateral load

R Array which generally stores load parameters

RES Plate compressive load parameter,  $\sigma_r/\sigma_{yp}$

where

$\sigma_r$  = plate compressive residual stress

$\sigma_{yp}$  = (see QIR)

RESI	Increment in "loop" of RES
RESN	Number of increments in "loop" of RES
REY	Square root of plate yield stress
RIN	Storage for important plate strain values
RMAX	Maximum plate strain
RX	Input value of increment of plate strain, $\frac{\Delta \epsilon_e}{\epsilon_{cr}}$ where $\epsilon_e$ is the increment of plate strain $\epsilon_{cr} = \frac{\sigma_{cr}}{E}$
R1I	Increment in "loop" of axial load parameter R(1) where $R(1) = P/P_{cr}$ $P$ = axial load $P_{cr}$ = (see CD)
R1N	Number of increments in "loop" of R(1)
R4I	Increment in "loop" of stiffener yield stress
R4N	Number of increments in "loop" of stiffener yield stress
R5I	Increment in "loop" of plate yield stress
R5N	Number of increments in loop of stiffener yield stress
R8	Plate strain increment used only in the range of high moment values in positive bending portion of m- $\theta$ -P curves
R8A	Plate strain increment generally used
SN	Sine of slope angle
SIFN	Stiffener flange strain corresponding to maximum moment capacity
TLEN	Maximum ultimate panel length, or storage location for increment in chord length

VAL1, VAL2, Dummy variables in subroutine VAL  
VAL3, VAL4

XLEN Maximum ultimate panel length, or storage location  
for increment of center deflection

The following variables and arrays are intermediate and have no general  
definition

BK	FIX2	T
C	FS	V
C1	FST	W
C2	F1	X*
C3	F2	Y*
C4	F3	Z1
C5	GM	Z2
C13	HS	ZK
C14	FSX3	ZL
F	RPL	ZM
FIX1	SP	ZN

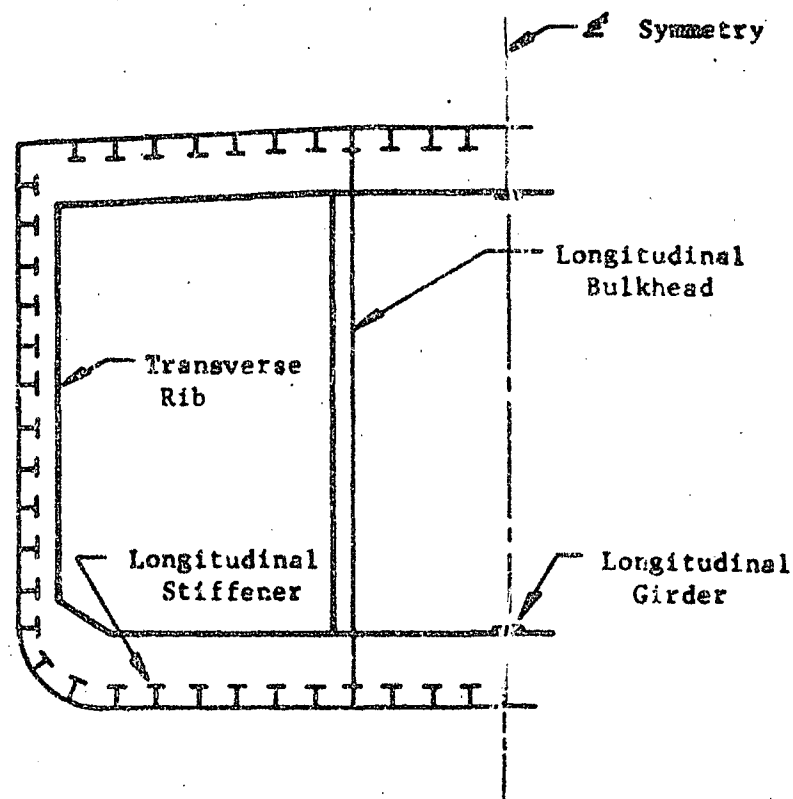
---

\*Array

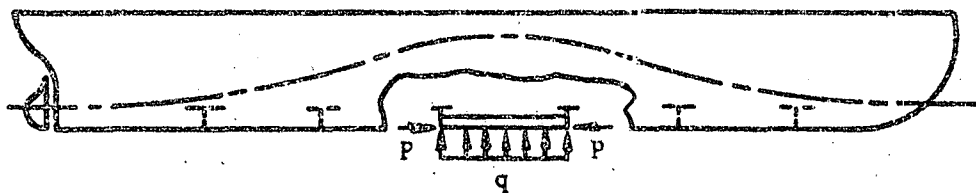
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FIGURES



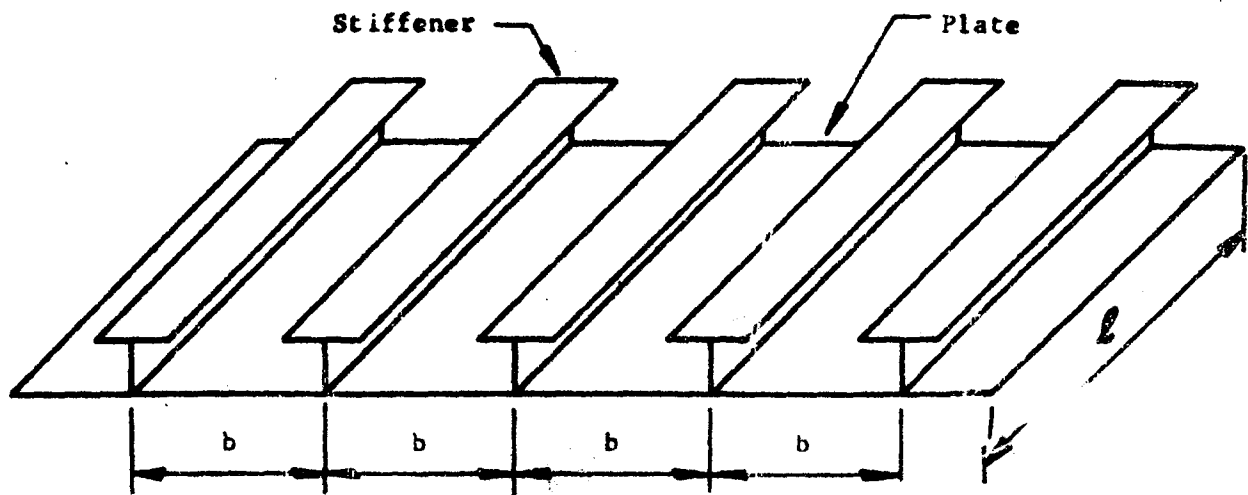


(a) TYPICAL MID-SHIP CROSS SECTION

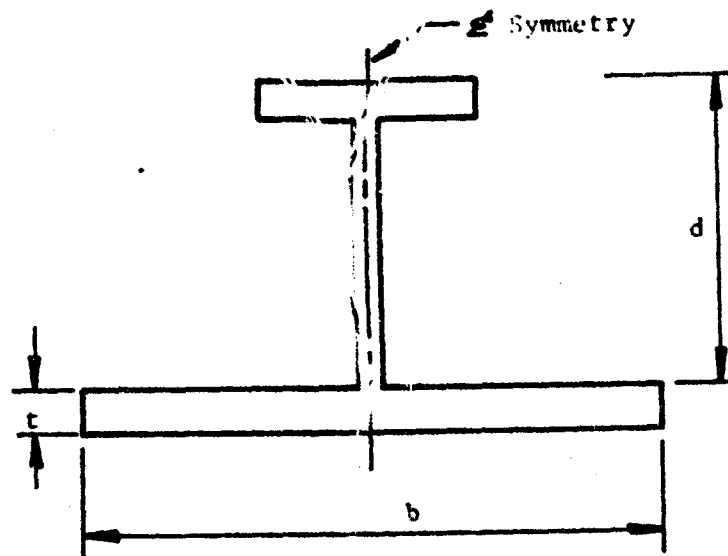


(b) LOADING ON THE SHIP BOTTOM PANEL DUE TO WAVE ACTION-HOGGING

Fig. 1 LONGITUDINALLY STIFFENED PLATE PANELS IN THE SHIP BOTTOM STRUCTURE.



(a) TYPICAL CROSS SECTION OF A SHIP HULL



(b) TYPICAL SECTION USED BY COMPUTER PROGRAM

Fig. 2 CROSS SECTIONS BEING ANALYZED

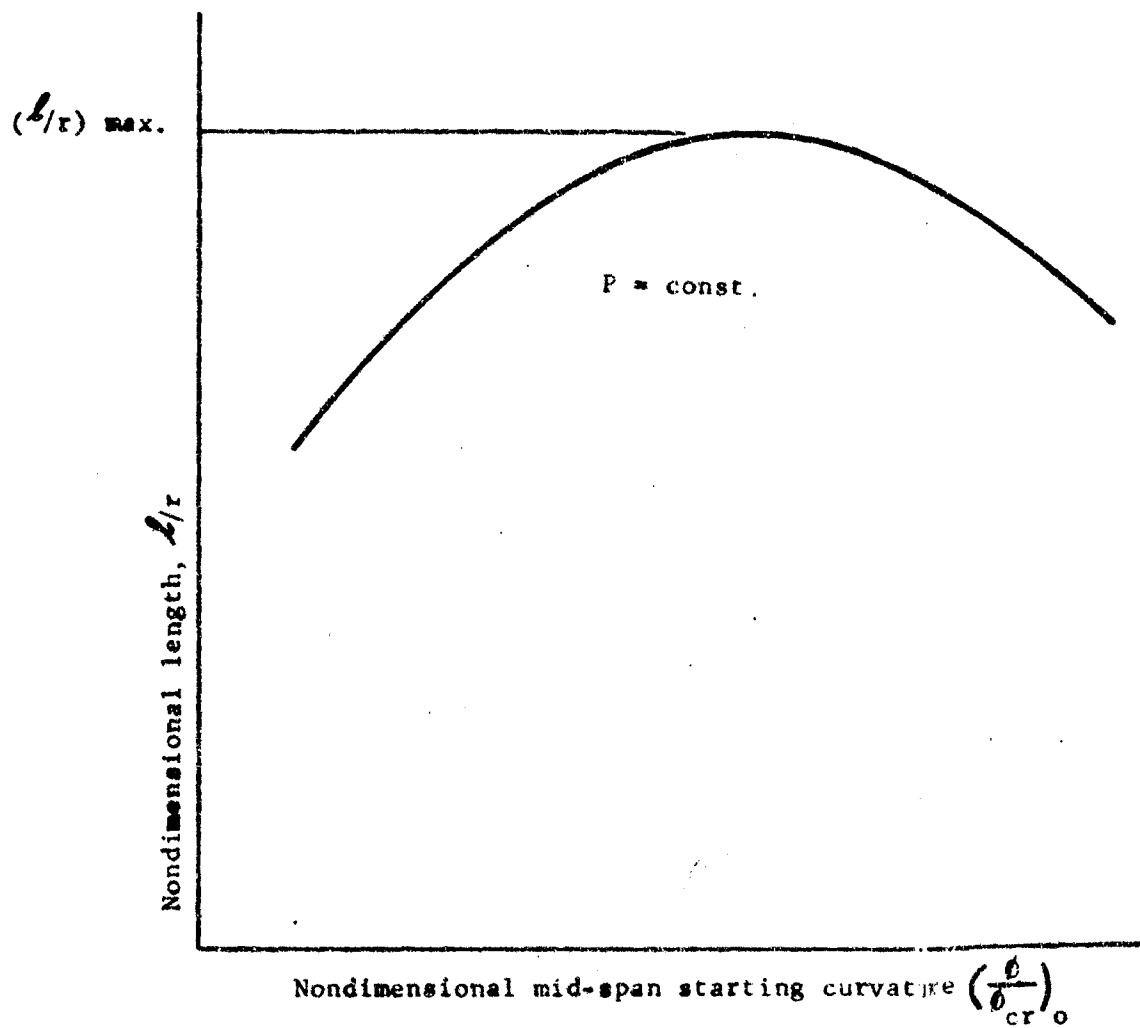


Fig. 3 DETERMINATION OF MAXIMUM LENGTH

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APPENDIX

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Subroutine INTEG	A10
Subroutine VALUES	A17
Function CUR	A18
Subroutine PARAM	A19
Function VAL	A20
Function EC	A21
Example Run	A22

# MAIN PROGRAM

PAGE

A-2

THIS IS THE MAIN PROGRAM. IT COMPUTES A MOMENT-CURVATURE CURVE FOR A GIVEN AXIAL LOAD.

THE FOLLOWING QUANTITIES ARE PLACED IN COMMON SO THAT MORE SPACE IS LEFT AVAILABLE FOR THE STORAGE OF VALUES USED IN THE COMPUTER PROGRAMS

```
COMMON R(20),O(10),P(30),F(120),CH(200),LRUN,NSUM,X(25),Y(30),RES
COMMON R,Q(1),OLIC,DSI,FIC,AST,AFS,CMX1,CMX2,PNT1
COMMON NN,NRUN,R4N,R4I,NR4,R5N,R5I,NR5,ASTN,ASTI,NAST,AFSN,AFRI
COMMON NAFS,RESN,RESI,NRES,RN,RI,NWB,RIN,R1I,NRI,RMAX,PSX1,SYN
COMMON PSX2,RPL,V,W,T,J,DEFA,LL,PSX3,RIN,L,TALL
COMMON C,INUM,REY,EY,CP,PHC,OIP,JA,IR,IC,ID,IE
COMMON DST,SN,CS,JA,C1,C2,C3,C4,C5,C11,PHC1,C13,C14,FMON,CA,CC,CD
COMMON TIEN,XIEN,EFIX,EPIN
COMMON LT,TALLA,OL1A,OLICA,NSUMA,F1,RK,F2,F3,GH,FS,71,72,73,74,75
```

READ NUMBER OF CYCLES

```
READ 55,NN
65 FORMAT(13)
II=1
```

READ RUN NUMBER (NRUN) AND NUMBER OF LATERAL LOAD BEING RUN (SET NO., IALL)

```
19 READ 20,NRUN,IALL
20 FORMAT(15,12)
LRUN=NRUN-1
```

READ MODULUS OF ELASTICITY (K.S.I.)

```
74 READ 75,R(4)
75 FORMAT(7.0)
```

READ STRAIN INCREMENT, RATIO OF CRITICAL PLATE STRAIN INEG. REFD.

```
READ 75,RX
```

READ INCREMENT OF SEGMENT LENGTH (NON-DIMENSIONALIZED BY MULTIPLICATION BY REY/R, WHERE R IS THE RADIUS OF GYRATION AND REY IS THE SQUARE ROOT OF THE PLATE YIELD STRAIN), INCREMENT OF MID-HEIGHT CURVATURE (NON-DIMENSIONALIZED BY DIVISION BY THE CRITICAL PLATE CURVATURE)

```
READ 145,DSI,FIC
145 FORMAT(2F7.4)
```

READ LATERAL LOAD (C/D/T,0 IN P.S.I.) (LOOP .I.E. INITIAL INCREMENT, NO. OF VALUES), SIGNAL OF MORE LATERAL LOADS, LOOP OF POSSIBLE ADDITIONAL LATERAL LOADS

```
READ 116,Q(1),OLIC,NSUM,I,T,TALLA,OL1A,OLICA,NSUMA
116 FORMAT(2F7.2,3I2,2F7.2,12)
```

READ YIELD STRESS OF STIFFENER (K.S.I.) (LOOP)

READ 115.R4N,R4I,NR4  
115 FORMAT(2F7.2,12)

READ YIELD STRESS OF PLATE (LOOP) (K.S.I.)

READ 115.R5N,R5I,NR5

READ AREA RATIO(STIFFENER TO PLATE) (LOOP)

READ 115.ASTN,ASTI,NAST

READ AREA RATIO (FLANGE TO STIFFENER) (LOOP)

READ 115.AFSN,AFSI,NAFS

READ RESIDUAL STRESS IN PLATE (AS RATIO OF PLATE YIELD STRESS)  
(LOOP)

READ 115.RESN,RESI,NRES

READ B/T (LOOP)

READ 115.BN,BI,NBR

READ AXIAL LOAD (AS RATIO OF CRITICAL LOAD) (LOOP)

READ 115.R1N,R1I,NP1

BEGIN LOOPING PARAMETERS

R(4)=R4N-R4I

DO 2270 J R4=1,NR4

R(4)=R(4)+R4I

R(5)=R5N-R5I

DO 2270 J R5=1,NR5

R(5)=R(5)+R5I

AST=ASTN-ASTI

DO 2270 J AST=1,NAST

AST=AST+ASTI

AFS=AFSN-AFSI

DO 2270 J AFS=1,NAFS

AFS=AFS+AFSI

RES=RESN-RESI

DO 2270 J RES=1,NRES

RES=RES+RESI

R=RN-BI

DO 2270 J BR=1,NBR

R=P+R

R(1)=R1N-R1I

DO 2270 J R1=1,NR1

R(1)=R(1)+R1I



## COMPUTE CROSS-SECTION VALUES

```

P(1)=(1.+AST)/(1.-AFS)
Q(1)=AST*AFS*(1./(1.+AST))
Q(2)=AST/P(1)
Q(3)=1./(1.+AST)
P(2)=Q(1)+Q(2)
Q(4)=P(2)-0.5*Q(2)
P(3)=1.-Q(4)
Q(5)=(Q(1)/Q(4)+2.-3.*Q(4))/3.
PRINT 62

```

```

62 FORMAT(1H1.43X,32H ***MOMENT CURVATURE RELATION***//)

```

```

LRUN IS THE COUNTER OF THE RUN BEING EXECUTED

```

```

LRUN=LRUN+1
PRINT 15,LRUN

```

```

15 FORMAT(52X,11HRUN NUMBER ,15.//)

```

```

PRINT X-SECTION PROPERTIES

```

```

PRINT 180,AST,AFS,R,Q(1),Q(2),Q(3),Q(4),Q(5)
180 FORMAT(5X,4HAST=,F8.6,2X,4HAFS=,F8.6,3X,4HPR/T=,F7.3,      3X,3HAP=,F
18.6,3X,3HAW=,F8.6,3X,3HAP=,F8.6,3X,3HCT=,F8.6,3X,3HSM=,F8.6//)

```

```

P(4)=0.27661*R/R(4)
CRS=RES+R(5)+P(4)
P(10)=P(4)+R(4)
R(11)=P(4)+Q(5)
CB=CRS/(CRS+R(11))

```

```

PARAM ADJUSTS THE STRAIN INCREMENT FOR THE CROSS-SECTION BEING RUN

```

```

R(R)=RY
CALL PARAMS,PAR,R(1)

```

```

COMPUTE MAX. PLATE STRAIN

```

```

IF (R(11)-1.1252,252.259
252 R(12)=R(11)
RMAX=R(11)
GO TO 261
259 R(7)=1.
260 R(12)=1.2+R(7)*0.6-0.45*R(7)+0.2+0.45*R(7)*1-0.21
R(16)=0.4*(1.+(1.-CB)*(1.-CB)-1.)*R(7)+1.5*(1.+(1.-CB)*(1.-CB))*R(
112)
IF (R(16)-R(11))3,5,5
3 R(7)=R(7)+R(R)
GO TO 260
5 RMAX=R(7)

```

```

COMPUTE LOCATION OF NEUTRAL AXIS (PNT1) AND MAX. PLATE STRESS AND
MAX. MOMENTS RESP. (PSX1 AND CMX1 FOR POS. BEND., PSX2 AND CMX2 FO
R NEG. BEND.)

```

```
261 P(5)=Q(3)*R(12)-Q(4)*R(11)
    PNT1=0.5*(R(11)+R(10)+P(2)-P(5))/R(10)+Q(2)
    IF(PNT1)275,267,267
267 IF(PNT1-1.1268,268,271
268 CMX1=(2.*PNT1+Q(4)-0.5*PNT1*R(10)+Q(2)+Q(4)+R(10)+Q(3)+P(5))/Q(5)
    PSX1=R(12)
    GO TO 280
271 STFN=(-Q(2)+R(10)-P(5)+R(11)/Q(1)
    PSX1=(R(11)-Q(3)+STFN-R(10)+Q(2))/Q(3)
    CMX1=(P(3)+R(10)-STFN+Q(1)-Q(4)*(R(10)+Q(3)-P(5)))/Q(5)
    GO TO 280
275 PSX1=(R(11)+R(10)+P(2))/Q(3)
    CMX1=Q(4)+R(10)+PSX1+Q(3)/Q(5)
280 PSX2=(R(11)-R(10)+P(2))/Q(3)
    CMX2=-Q(4)+R(10)-PSX2+Q(3)/Q(5)
    IF(CMX1)338,308,310
308 PRINT 309,CMX1
309 FORMAT(25X, 62HNO POSITIVE BENDING MOMENT CAPACITY
    1 (CMX1 IS NEGATIVE) CMX1=,F9.4)
    GO TO 2270
310 R(3)=P(2)+R(10)+P(5)
    PRINT 330,R(3),R(10),R(11),R(12),RMAX
330 FORMAT( 5X,9HMAX LOAD=,F9.6,2X,12HSTIFF YIELD=,F9.6,2X,12HPLATE
    1YIELD=,F9.6,2X,12HMAX PL STRS=,F9.6,2X,12HMAX PL STRN=,F9.6//)
    PRINT 333,PSX1,CMX1,PNT1,PSX2,CMX2
333 FORMAT( 9X,5HPSX1=,F9.6,8X,5HCMX1=,F9.6,8X,5HPNT1=,F9.6,8X,5HPNT2
    1=,F9.6,8X,5HCMX2=,F9.6//)
    IA=1
```

N LESS THAN 100 FOR NEG. BEND.  
N GREATER THAN 100 FOR POS. BEND.

```
    N=100
    R(7)=R(11)
    GO TO 2050
360 PIN=R(7)
    PRINT 370,R(11),RES,CR,R(8)
370 FORMAT(17X,11HAXIAL LOAD=,F6.3,4X,20HPLATE RESID. STRESS=,F5.3,4X,
    14HC/R=,F5.3,4X,17HSTRAIN INCREMENT=,F6.3//)
    PRINT 365,R(6),R(5),R(4)
365 FORMAT(15X,19HMOD. OF ELASTICITY=,F7.1,5X,19HPLATE YIELD STRESS=,F
    17.3,6X,20HSTIFF. YIELD STRESS=,F7.3,///)
    PRINT 390
390 FORMAT( 39X,6HSTRAIN,6X,9HCURVATURE,7X,6MMOMENT,7X,(1HN,3X,1HJ)
    1A=2
391 GO TO (360,392),IA
392 L=1
```

COMPUTE VALUES FOR MOMENT AND CURVATURE EQUATIONS

```
400 P(6)=R(7)-R(11)
    P(7)=R(7)+R(10)
    P(8)=R(7)-R(10)
    P(9)=R(7)+R(11)
```

```

440 P[10]=R[7]-R[11]
    P[11]=P[7]+P[2]
    P[12]=P[8]+P[2]
    P[13]=P[9]+Q[3]
    P[14]=P[13]+Q[3]
    P[15]=0.5*Q[2]
    P[16]=P[15]+Q[4]+P[7]*P[7]
    P[17]=P[15]+Q[4]+P[8]*P[8]
    P[18]=4.*P[15]+R[7]+R[10]*Q[4]

```

```

C
C
C
C
C
COMPUTE CURVATURES FI(N) AND MOMENTS CM(M)
J=1 TO 10 IS FOR NEG. BENDING
J=11 TO 16 IS FOR POS. BENDING

```

```

    RPL=R[7]+CRS
    FIX1=Q[4]+ARSF(P[8])
    IFIN=1001540,550,1070
550 IFIARSF(R[1]-RMAX1540,560,790
560 IFIARSF(R[7]-P[10])570,570,870
570 IFIRPL=-1.1540,540,700
580 J=1
    V=P[6]
600 FI(N)=V
610 IFIARSF(FI(N))-FIX11620,620,630
620 GO TO 11430,2070,1430,2070,1430,2070,1430,2070,1430,2070,1430,2070,J
630 GO TO 1640,1430,750,1430,830,1430,950,1430,1030,1430,J
640 J=2
    V=P[6]-P[12]
    W=P[17]
670 IFIV12070,2070,480
680 FI(N)=-W/V
    GO TO 610
700 J=3
710 R[15]=1.2*RPL**0.6-0.65*RPL**0.2+0.45*RPL**(-0.2)
711 P[15]=R[15]-CRS
715 P[19]=(R[15]-P[7])*Q[3]+Q[4]
    IFIJ-41730,730,1220
730 V=P[6]+P[10]/Q[4]
740 IFIV1500,600,2070
750 J=4
    V=V-P[12]
    W=P[17]
    GO TO 670
790 P[30]=CRS+Q[3]
    IFIAPSF(R[1]-P[10])1800,990,990
800 J=5
    V=P[6]-P[13]-P[30]
    GO TO 740
830 J=6
    V=V-P[12]
    W=P[17]
    GO TO 670
870 J=7
    V=P[6]
    W=-P[16]

```

```

900 IF(W)910,600,910
910 T=V*V+4.*4
    IF(T)2070,930,930
930 FI(N)=0.5*(V+SQRTF(T))
    GO TO 610
950 J=8
    W=-P(1A)
    V=P(6)-P(12)
    GO TO 670
990 J=6
    V=P(6)-P(13)-P(30)
    W=-P(16)
    IF(V)900,2070,2070
1030 J=10
    V=V-P(12)
    W=-P(18)
1060 GO TO 670
1070 J=13
    FIX2=P(4)+P(7)
    IF(RP_-RMAY)1080,1080,1077
1077 R(15)=R(12)-CRS
    GO TO 715
1080 IF(RP_=-1.)1090,1090,710
1090 J=11
    V=P(6)
1110 FI(N)=V
1120 K=J-10
    IF(-FIX1-FI(N))1130,1130,700
1130 IF(FI(N)-FIX2)1140,1140,1150
1140 GO TO 11437,2140,1437,2140,1437,2140,K
1150 GO TO 11160,1437,1260,1437,1390,1437,K
1160 J=12
    V=P(6)-P(11)
    W=P(15)
1190 IF(V)1200,2140,2140
1200 FI(N)=-W/V
    GO TO 1120
1220 IF(R(7)-R(10))1240,1240,1300
1240 V=P(6)+P(19)/O(4)
    GO TO 1110
1260 J=14
    V=V-P(11)
    W=P(16)
    GO TO 1107
1300 J=15
    V=P(6)+P(19)/O(4)
    W=-P(17)
    IF(V)2140,1340,1340
1340 IF(W)1350,1110,1350
1350 T=V*V+4.*4
    IF(T)2140,1370,1370
1370 FI(N)=0.5*(V+SQRTF(T))
    GO TO 1120
1390 J=16
    V=V-P(11)

```

```

W=P[18]
GO TO 1100
1430 IF(N-100)1432,1540,1432
1432 IF(L-5)1435,1540,1540
1435 DEFA=FI(N+1)-FI(N)
GO TO 1438
1437 DEFA=FI(N)-FI(N-1)
1438 IF(N-100)1440,1540,1440
1440 IF(DEFA-MS)1441,1540,1460
1441 IF(J-10)2070,2070,2140
1460 IF(DEFA-0.4)1540,1540,1470
1470 IF(J-10)1500,1500,1520
1500 R(7)=R(7)+0.3*P(8)
GO TO 1530
1520 R(7)=R(7)-0.3*R(8)
1530 L=L+1
IF(L-3)1400,400,1540
1540 R(13)=R(7)-FI(N)/Q(4)
P(20)=Q(4)+(R(13)-R(7))/Q(3)
IF(RPL-RMAX)1555,1555,1552
1552 R(15)=R(12)-GRS
1555 P(24)=Q(4)+Q(3)*(R(15)-R(13))
P(21)=0.5*(1./3.-Q(4))*(R(13)-R(7))+Q(2)
IF(FI(N))1600,1580,1600
1580 CM(N)=0.
GO TO 2220
1600 IF(J-10)1610,1610,1660
1610 Q(6)=3/41*P(7)/FI(N)
Q(8)=1.-Q(4)*P(8)/FI(N)
P(22)=(R(13)-R(10))+(P(3)+Q(1)+Q(8)*(P(3)-Q(8)/3.))+P(15)
P(23)=Q(6)+P(15)+P(7)+Q(4)-Q(6)/3.
P(27)=(R(13)+R(11))+Q(3)+Q(4)
GO TO 1660,1680,1700,1720,1740,1760,1780,1800,1820,1840,J

```

# NEGATIVE MOMENT EQUATIONS

```

1660 CM(N)=(-P(20)+P(21))/Q(5)
GO TO 2220
1680 CM(N)=(-P(20)+P(21)+P(22))/Q(5)
GO TO 2220
1700 CM(N)=(P(24)+P(21))/Q(5)
GO TO 2220
1720 CM(N)=(P(24)+P(21)+P(22))/Q(5)
GO TO 2220
1740 CM(N)=(-P(27)+P(21)+P(30))/Q(5)
GO TO 2220
1760 CM(N)=(-P(27)+P(21)+P(22)+P(30))/Q(5)
GO TO 2220
1780 CM(N)=(-P(20)+P(21)-P(23))/Q(5)
GO TO 2220
1800 CM(N)=(-P(20)+P(21)-P(23)+P(22))/Q(5)
GO TO 2220
1820 CM(N)=(-P(27)+P(21)-P(23)+P(30))/Q(5)
GO TO 2220
1840 CM(N)=(-P(27)+P(21)+P(22)-P(23)+P(30))/Q(5)

```

GO TO 2220  
 1860  $O(7) = 2(41 + P(8)) / F(11)$   
 $O(9) = 1. - O(4) + P(7) / F(11)$   
 $P(25) = (R(13) + P(10)) + (P(3) * O(1)) + O(9) * (P(3) - O(9) / 3.) * P(15)$   
 $P(26) = O(7) + P(15) + P(8) + (O(4) - O(7) / 3.)$   
 GO TO 1920, 1940, 1960, 1980, 2000, 2020, K

# POSITIVE MOMENT EQUATIONS

1920  $CM(1) = (-P(21) + P(21)) / O(5)$   
 GO TO 2220  
 1940  $CM(N) = (-P(21) + P(21) + P(25)) / O(5)$   
 GO TO 2220  
 1960  $CM(N) = (P(24) + P(21)) / O(5)$   
 GO TO 2220  
 1980  $CM(N) = (P(24) + P(21) + P(25)) / O(5)$   
 GO TO 2220  
 2000  $CM(N) = (P(24) + P(21) - P(26)) / O(5)$   
 GO TO 2220  
 2020  $CM(N) = (P(24) + P(21) + P(25) - P(26)) / O(5)$   
 GO TO 2220  
 2030  $N = N - 1$   
 IF  $(N - 2110, 2110, 2070)$   
 2042 IF  $(0.99 - CM(N + 1) / CMX2, 12080, 2080, 2050)$   
 2050  $LL = 1$   
 2051  $RPL = R(7) + CRS$   
 IF  $(RPL - 1., 12053, 2053, 2067)$   
 2053  $PSX3 = R(7)$   
 2054 IF  $(PSX3 - PSX2, 12060, 2060, 2055)$   
 2055  $FST = R(1) - PSX3 + O(3)$   
 IF  $(FST / P(21) - R(10), 1301, 2270, 2270)$   
 2060 GO TO (2061, 2064), 1A  
 2061  $R(7) = R(7) + R(8)$   
 GO TO 2051  
 2064  $P(7) = R(7) + 0.1 * R(8)$   
 $LL = LL + 1$   
 IF  $(LL - 10, 12051, 2051, 3001)$   
 2067  $PSX3 = 1.2 * RPL + 0.6 - 0.65 * RPL + 0.2 + 0.45 * RPL + (-0.2)$   
 $PSX3 = PSX3 - CRS$   
 GO TO 2054  
 2070  $R(7) = R(7) + R(8)$   
 GO TO 2042  
 2080  $FI(N) = FI(N + 1) - 1.$   
 $CM(N) = (CMX2 - CM(N + 1)) + (FI(N) - FI(N + 1)) / (-500. - FI(N + 1) + CM(N + 1))$   
 GO TO 2220  
 2110  $N = 100$   
 $R(7) = R(1)$   
 2130  $N = N + 1$   
 IF  $(N - 2001, 2140, 2241)$   
 2140  $R(7) = R(7) + P(8)$   
 IF  $(0.99 - CM(N - 1) / CMX1, 12152, 2152, 2160)$   
 2152  $FI(N) = FI(N - 1) + 1.$   
 $CM(N) = (CMX1 - CM(N - 1)) + (FI(N) - FI(N - 1)) / (500. - FI(N - 1) + CM(N - 1))$   
 GO TO 2220  
 2160 IF  $(PNT1, 12161, 2161, 301)$

```

2161 LL=1
2180 RPI=R(7)+CRS
      IF(RP_-1.12182,2182,2187)
2182 PSX3=R(7)
2183 IF(PSX1-PSX312184,2184,391)
2184 R(7)=R(7)-0.1*R(8)
      LL=LL+1
      IF(LL-1012180,2180,3001)
3001 PRINT 3002
3002 FORMAT(1X,27HEXIT CALLED. II EXCEEDS 10. 1
      GO TO 2270
2187 PSX3=1.2+RPI**0.6-0.65*RPL**0.2+0.45*RPL**(-0.2)
      PSX3=PSX3-CRS
      GO TO 2183
2220 PRINT 2230,R(7),F1(1),CM(1),N,J
2230 FORMAT(36X,F9.3,2(4X,F11.6),3X,I3,2X,I2)

```

IF FOR N GREATER THAN 100 THE MOMENT IS NEAR THE MAXIMUM NEGATIVE  
MOMENT, CMX2, USE A LOWER STRAIN INCREMENT (R(8)) SO THAT MORE  
VALUES OF POSITIVE BENDING CAN BE COMPUTED.

```

      IF(IN-10012030,2030,2231)
2231 IF(CMIN1-1.85+CMX212232,2233,2233)
2232 R(8)=RRA
      GO TO 2130
2233 R(8)=RR
      GO TO 2130
2241 CALL INTFG
2270 CONTINUE
2280 II=II+1
      IF(II-NN110,19,3000)
3000 CALL EXIT
      END

```

# SUBROUTINE INTEG

2100  
A 20

GIVEN A MOMENT-CURVATURE RELATION FOR A SECTION, SUBROUTINE INTEG  
WILL DETERMINE THE MAXIMUM FIXED AND PINNED LENGTHS THAT THE  
SECTION CAN SUSTAIN UNDER A GIVEN AXIAL AND LATERAL LOADING PAIR

## SUBROUTINE INTEG

THE FOLLOWING QUANTITIES ARE PLACED IN COMMON SO THAT MORE STORAGE  
SPACE IS LEFT AVAILABLE FOR THE COMPUTER PROGRAMS

```
COMMON R(20),Q(10),P(30),F(120),CM(200),LRUN,NSUM,X(25),Y(30),RES  
COMMON R,Q(1),DLIC,DST,FIC,AST,AFS,CMX1,CMX2,PSX1  
COMMON NN,NRUN,R4N,R4I,NR4,R5N,R5I,NR5,ASTI,ATI,NAST,AFSN,AFSI  
COMMON NAFS,RFSN,RESI,NRES,RN,RI,NRB,RIN,RII,NRI, PMAX,PSX1,STFN  
COMMON PSX2,PPL,V,W,T,J,DEFA,LI,PSX3,RIN,L,I,LI  
COMMON C,INUM,REY,EY,CR,PHC,QIP,JA,IR,IC,ITE  
COMMON DST,SN,CS,JR,C1,C2,C3,C4,C5,C11,PHC1,C13,C14,FMOM,CA,CC,CD  
COMMON TIEN,XIEN,EFIX,EPIN  
COMMON IT,IALIA,Q1A,Q1CA,NSUMA,F1,RK,F2,F3,GH,FS,71,7J,7K,7L,7M  
COMMON FOI,DSII,FII,IS,IAFLI,OFL1,QF1IC,QFSUM
```

90 PRINT 100

100 FORMAT(1H1.44X,32H\*\*MAXIMUM SLENDERNESS RATIOS\*\*//)

PRINT 130,LRUN

130 FORMAT(52X,11HRUN NUMBER ,15,//)

THE FOLLOWING LOOP IS TO CONVERT MOMENTS AND CURVATURES TO A FORM  
THAT THEY ARE NON-DIMENSIONALIZED BY DIVISION BY PLATE YIELD  
VALUES RATHER THAN CRITICAL BUCKLING VALUES

DO 1 I=1,200

CM(11)=CM(11)/R(11)

1 F(11)=F(11)/R(11)

CMX2=CMX2/R(11)

CMX1=CMX1/R(11)

## MATERIAL PROPERTIES

EY=R(51)/R(61)

REY=SQRT(EY)

## SECTION PROPERTIES

CR=R/V WHERE R IS THE RADIUS OF GYRATION AND V IS THE DISTANCE  
FROM THE PLATE TO THE ELASTIC NEUTRAL AXIS

CR=SQRT((1.-2.\*Q(41)\*Q(1))+1./3.-Q(41)\*Q(2)+Q(41)\*Q(41)/Q(41)

Q1L IS THE LATERAL LOAD (P.S.I.)

II IS A COUNTER OF LATERAL LOADS

LS=LT

IAFLI=IALI

OFL1=Q(1)

OFLIC=Q(1C)

NFSUM=NSUM

PRINT 120,DST,FIC



```

120 FORMAT(39X,13H*INPUT*,4HDSI=F8.6,5X,4HFIC=F8.6,11H* A-
C=D.001*D(4)*D(3)/(FY*R(5))
285 QLL=Q.L-QLYC
II=1
5010 QLL=Q.L+QLYC

```

SUBROUTINE VALUES ADJUSTS THE INCREMENTS OF SEGMENT LENGTH AND MID-HEIGHT CURVATURE FOR THE CROSS-SECTION BEING RUN

```

CALL VALUES (QLI,FICA,DSIA)
FOI=FIC-FICA
DSII=DSI-DSIA
FII=FOI/R(11)

```

ESTIMATE INITIAL MID-HEIGHT CURVATURE PHC

```

PHC=CJR(QLI)
PHC1=PHC-R(11)
EPYN=0.
EFIX=0.

```

```

INUM=II+1411-1
QIR=Q.L+C
JA=20

```

```

PRINT 5086,IRUN,INUM,QLL,QIR,PHC1,DSII,FOI
5086 FORMAT(12X,11HCASE NUMBER,15,1H.,12,4X,4HOLI=F6.4,4X,4HDIR=F6.
1,4X,54PHCI=F6.4,4X,5HDSII=F6.4,4X,4HFII=F6.4)
PRINT 5100

```

```

5100 FORMAT(//8X,4HCURV,12X,3HL/R,11X,3HX/R,12X,3HY/R,12X,3HMM,11X,3H
1LOPE,12X,1HP,10X,4HENDS)

```

INITIALIZE THE VALUES

```

5110 DO 5120 K=1,30
5120 Y(K)=0.

```

```

IA=1
IB=2
IC=1

```

```

5160 DO 5170 K=1,16
5170 X(K)=0.

```

```

ID=1
IE=1
X(17)=R(1)/R(11)
X(19)=DSII
DST=DSII
SN=0.
CS=1.
N=99

```

```

5260 N=N+1

```

```

IF(200-N)5275,5270,5270

```

```

5275 PRINT 5276

```

```

5276 FORMAT(1X,42HCURVATURE OUTSIDE LIMITS OF M-PHI-P CURVE
GO TO 448)

```

CHOOSING A CURVATURE THAT MATCHES THE COMPUTED ONE

```

5270 IF(FI(N)-PHC)5260,5280,5280
5280 F=(FI(N)-PHC)/(FI(N)-FI(N-1))
X(11)=CM(N)-F*(CM(N)-CM(N-1))
IF(X(11))5310,5310,5321

```

```

5310 PHC=PHC+0.05
GO TO 5270

```

```

5321 X(23)=PHC
JB=36

```

```

5323 JB=JB-1

```

```

IF(JB)5329,5330,5330
5325 Y(4)=1.
PRINT 5324

```

```

5326 FORMAT(3A4 ERROR**TOO MANY LENGTH INCREMENTS
GO TO 6163

```

```

5330 I=1
X(20)=0ST

```

FOR A FIRST GUESS ASSUME THAT THE CURVATURE AT THE TERMINAL END IS THE SAME AS THE CURVATURE AT THE INITIAL END.

```

X(25)=X(23)

```

CALCULATE THE VALUES OF THE FORCES AT THE INITIAL END AND CHORD LENGTH, CENTER DEFLECTION, AND MOMENT AT THE TERMINAL END.

```

5333 C1=(X(23)/4.+X(25)/4.)*CR*X(20)

```

```

C2=(CS-C1*REY*SN)*X(20)

```

```

C3=(SV/REY+C1*CS)*X(20)

```

```

C13=X(15)

```

```

C14=X(17)

```

```

X(12)=X(11)-(C2+(C13+QIR*0.5+C2)+C3*(C14/CR+C3*EV*0.5+QIR))

```

```

TLEN=C2

```

```

YLEN=C3

```

EVALUATE THE CURVATURE CORRESPONDING TO THE CALCULATED MOMENT BY USING A LINEAR INTERPOLATION

```

5400 N=N+1

```

```

IF(200-N)5520,5420,5420

```

```

5420 IF(CM(N)-X(12))5400,5430,5430

```

```

5430 IF(X(12)-CM(N-1))5440,5590,5590

```

```

5440 N=N-1

```

```

IF(N-1)5440,5460,5460

```

```

5460 C1=X(12)-CM(N)

```

```

IF(C1)5490,5490,5490

```

```

5490 GO TO (5540,5620),IN

```

```

5490 C1=X(11)-CM(N)

```

```

X(20)=0.5*X(20)+C1/(X(11)-X(12))

```

```

GO TO 5570

```

```

5520 C1=CM(200)-X(11)

```

```

N=199

```

```
X(20)=0.2*Y(20)+C1/(X(12)-X(11))
GO TO 5570
5560 Y(20)=0.1*Y(20)
5570 I=I+1
      IF(I-15)5333,5333,5700
5590 F=(CM(N)-X(12))/(CM(N)-CM(N-1))
      X(24)=F*(N)-F*(F*(N)-F*(N-1))
      GO TO 5650
5620 C4=(CM(1)-CM(2))/C1
      C5=C4*C1/(CM(2)-CM(1))
      X(24)=(C4+1)*F(1)-SQRT(F(1)*(1-F(1))-F(2)*C5)
```

C  
C  
C  
C  
CHECK THE CONVERGENCE OF THE ASSUMED AND CALCULATED CURVATURES.  
IF THEY ARE VERY CLOSE CONTINUE ON, IF NOT, RECALCULATE THE VALUES

```
5650 IF(ABS(F(1)-X(25)/X(24))-0.00001)*00,5660,5660
5660 I=I+1
      IF(I-15)5660,5660,5700
5680 X(25)=X(24)
      GO TO 5333
```

C  
C  
C  
C  
COMPUTE THE SLOPE AT THE TERMINAL POINT AND USE THIS TO COMPUTE  
THE OTHER TERMINAL VALUES

```
5700 C1=X(23)+X(24)
      X(14)=X(13)+0.5*C1+X(20)+CR*REY
      SN=SINF(X(14))
      CS=COSF(X(14))
      EMOM=SN-SINF(X(13))
      F=CS-COSF(X(13))
      X(16)=C13+0.7R*(C2-REY+EMOM/CR)
      X(18)=C14+0.7R*FY*(C3+CR+F)
      X(9)=X(8)+C3
      X(6)=X(5)+2.*C2
      X(3)=X(2)+2.*X(20)
      C5=X(24)-X(23)
      CA=X(19)
      CB=X(20)
      GO TO (5A30,6010),IF
5830 IF(X(12))5A40,6010,6010
```

C  
C  
C  
C  
C  
C  
COMPUTE THE VALUES FOR THE PINNED CASE. FUNCTION RC CALCULATES  
THE SEGMENT LENGTH CORRESPONDING TO ZERO MOMENT BY NEWTON'S METHOD  
THE PANEL LENGTH, CHORD LENGTH, CENTER DEFLECTION, AND SLOPE ARE  
INTERPOLATED LINEARLY.

```
5840 C1=(X(12)-X(11))*X(19)
      C2=(X(10)-X(11))*X(20)
      C4=(X(10)+X(20))*X(19)+X(20)
      C3=(C1+C2)/C4
      C4=(C1+X(10)-C2+X(20))/C4
5890 C1=BC(C3,C4,X(11),0.1)
      Y(29)=Y(13)+(X(23)+0.5+C5+C1/X(20))*C1+CR*REY
      Y(23)=0.
      IE=2
```

```

10=2
IF (C4*X1+C4X2) 15950, 5940, 5940
5940 DST=0.4*DST1
5950 Y(5)=Y(2)+2.*C1
      Y(11)=X(5)+C1*2.*TLEN/CR
      Y(17)=X(8)+C1*XLEN/CR
6010 IF (X(14)) 15020, 6170, 6170

```

COMPUTE THE VALUES FOR THE FIXED END CASE  
FUNCTION BC COMPUTES THE SEGMENT LENGTH CORRESPONDING TO ZERO  
SLOPE BY NEWTON-S METHOD. THE PANEL LENGTH, CHORD LENGTH, AND  
CENTER DEFLECTION ARE INTERPOLATED LINEARLY. THE END MOMENT IS  
COMPUTED BY A PARABOLIC EQUATION IN FUNCTION VAL.

```

6020 C1=0.5*C5*REY*CR/X(20)
      C2=X(23)+REY*CR
      C1=BC(C1,C2,X(13),0.1)
      Y(30)=0.
      Y(6)=X(2)+2.*C1
      Y(12)=Y(5)+C1*2.*TLEN/CR
      Y(18)=X(8)+C1*XLEN/CR
      Y(24)=VAL(X(10),X(11),X(12),C1,CA,CR)
6163 GO TO 16230, 6430, 1A

```

LET THE TERMINAL VALUES OF THE SEGMENT JUST COMPLETED BECOME INIT-  
IAL VALUES FOR THE NEW SEGMENT. USING THE NEW INITIAL VALUES, GO  
BACK AND CALCULATE THE VALUES FOR THE NEXT SEGMENT.

```

6170 DO 6180 1+1,23
      X(22)=0.
6180 Y(11)=X(1)+11
      I=1
      GO TO 5323

```

```

6230 IF (Y(5)-Y(4)) 6250, 6240, 6240
6240 GO TO 16430, 6510, 1C
6250 1A=2
      1B=1B-1
      K=6

```

COMPUTE THE ULTIMATE CONDITIONS AS DEFINED BY ZERO SLOPE IN THE  
PANEL LENGTH-DEFLECTION CURVE AND PRINT OUT THE RESULTS

FIND THE MAXIMUM LENGTH Y(K) AND THE CHANGE IN CURVATURE C(3) FROM  
THE PREVIOUS POINT THAT CORRESPONDS TO THIS LENGTH. USE A PARABOL-  
IC APPROXIMATION

```

6280 C1=Y(K)-Y(K-4)
      C2=Y(K-2)+Y(K-2)+Y(K-4)
      C3=-0.5*F(1)+C1/C2
      CA=F(1)
      CB=F(1)
      Y(K)=Y(K-2)-0.125*C1+C1/C2
      IF (Y(K-4)-0.00001) 6335, 6335, 6340

```

```

6335 IF (R-216338,6338,6336)
6336 PRINT 6337
6337 FORMAT(40H ERROR**FIXED END**INITIAL PHC TOO HIGH 1
GO TO 6340
6338 PRINT 6339
6339 FORMAT(30H ERROR**S.S. END**INITIAL PHC TOO HIGH 1

```

FIND THE VALUES OF CHORD LENGTH, CENTER DEFLECTION, MOMENT, AND SLOPE RESPECTIVELY THAT CORRESPOND TO THE POINT OF MAXIMUM MOMENT. USE A PARABOLIC APPROXIMATION.

```

6340 K=K+6
6352 Y(K)=VAL(Y(K-4),Y(K-2),Y(K),C3,CA,CB)
K=K+6
IF (K-31) 6352,6340,6360

```

FIND THE MID-HEIGHT CURVATURE (CC) AND AXIAL FORCE (CD) THAT CORRESPOND TO THE ULTIMATE CONDITION. BOTH ARE NON-DIMENSIONALIZED BY THE CRITICAL VALUES.

```

6360 CC=(P+C-F11+C3)*R(11)
6370 CD=R(11)+0.1P*FY*P(3)/0.41*(Y(K-18)+(COSF(Y(K-6))-.1)/C1)

```

PRINT ANSWERS FOR F.E. AND P.E. CURV., MOM. AND P HAVE BEEN NON-DIMENSIONALIZED BY DIVISION BY THE CORRESPONDING PLATE CRITICAL VALUES. THE SLOPE IS IN DIMENSIONAL FORM. L/R, X/R AND Y/R HAVE NOT BEEN FURTHER NON-DIMENSIONALIZED. CURV. AND Y/R ARE CURVATURE AND DEFLECTION AT MID-HEIGHT RESP. X/R REFERS TO THE CHORD LENGTH, AND L/R TO THE TOTAL LENGTH OF THE PANEL. MOMENT AND SLOPE REFER TO THE END VALUES.

```

TLEN =Y(K-30)/REY
YLEN =Y(K-24)/REY
FMOM=Y(K-12)*R(11)
IF (Y(K-6)-0.0000001) 6395,6396,6398
6396 FFIX=TLEN
PRINT 6307,CC,FFIX,YLEN,Y(K-18),FMOM,Y(K-6),CD
6397 FORMAT(7I5X,F10.6),4X,5HFIXED,///)
GO TO 6420
6398 EPIN=TLEN
PRINT 6309,CC,EPIN,YLEN,Y(K-18),FMOM,Y(K-6),CD
6399 FORMAT(7I5X,F10.6),5X,4HS.S.,////////)

```

```

6420 GO TO (6430,6480),IR
6430 IF (Y(5)-Y(3)) 6440,6510,6510
6440 K=5
IR=IR+1
IF (IR-2) 6440,6460,6451
6451 PRINT 6452
6452 FORMAT(21H ERROR**IR EXCEEDS 2 1
6460 IC=2
GO TO 6200
6480 PUNCH 6481,IRUN,INUM,EFIX,EPIN,R(1),AST,AFS,P,RES,R(5),R(4),R(4),0
1LL

```

6481 FORMAT I15, I2, 2F6.2, 2F4.2, 2F6.5, F5.1, F5.4, 2F5.1, F7.1, F5.11

CHECK TO SEE IF THE MAXIMUM LATERAL LOAD CASE HAS BEEN RUN, AND  
ALSO IF ADDITIONAL LATERAL LOADS ARE TO BE RUN

11=11+1  
IF(11-NSUM) 5010, 5010, 6490  
6490 LS=LS+1  
IF(LS-213000) 6491, 6580  
6491 IALL=IAIFA  
OL1=OL1A  
OLIC=OLIC4  
NSUM=VSIUMA  
GO TO 285

6510 JA=JA-1  
IF(JA) 6521, 6530, 6530  
6521 PRINT 6522  
6522 FORMAT I4X, 4A PT00 MANY PHC VALUES REQUIRED. USE A LARGER FIG.  
GO TO 6480

INITIALIZE THE VALUES. CHOOSE A NEW CENTER CURVATURE AND BEGIN  
CALCULATIONS TO AGAIN FIND THE CORRESPONDING PINNED AND FIXED  
LENGTHS.

6530 PHC=PHC+FIT  
DO 6560 K=1, 27, 2  
Y(K)=Y(K+21)  
6560 Y(K+1)=Y(K+31)  
GO TO 5160

6580 IALL=IAFIL  
OL1=OFL1  
OLIC=OFLIC  
NSUM=VFSUM  
3000 RETURN  
END

# SUBROUTINE VALUES

PAGE

A27

```
SUBROUTINE VALUES (OLL,FICA,DSIA)
COMMON R(20),Q(10),P(30),F(120),CH(200),LRHM,NSUM,X(25),Y(30),RFS
COMMON R,Q,I,OL,IC,DSI,FIC,AST,AFS,CHX1,CHX2,PNT1
COMMON NN,NRHM,R4N,R4I,NP4,R5N,R5I,NR5,ASTN,ASTI,NAST,AFRN,AFSI
COMMON NAFS,RFSN,RESI,NRES,RN,R1,NR,R1N,R1I,NR1,PMAX,PSX1,STEN
COMMON PSX2,RPL,V,M,T,J,DEFA,LI,PSX3,RIN,L,TALI
COMMON C,INIM,REF,EY,CP,PHC,DIR,JA,IR,IC,ID,IE
COMMON DST,SN,CS,JR,C1,C2,C3,C4,C5,C11,PHC1,C13,C14,ENOM,CA,CC,CD
COMMON TIEN,XIEN,FFIX,FPIN
COMMON IT,TALLA,OL1A,OL1CA,NSUMA,F1,RK,F2,F3,GH,FS,7I,7J,7K,7L,7M
FS=.0000106*Q(1)*OLL-.0007*Q(1)+1.9
ZJ=1.+0.0765*(R(4)-50.)
7K=1.+0.01*(R-85.)
7N=0.0086*(R(4)+0.6)
7L=1.+0.02*(R-85.)
7M=1.+2.5*(AST-.35)
SP=0.014*3*(0.0086*(R(4)+0.66)
IF(R(1)-SP)5,5,6
5 7I=1.
GO TO 7
6 7I=1.-(R(1)-SP)/(0.016*(R-.36)
7 DSIA=FS*7I*7N*7K*REF/0.04
FICA=7I*7L*7M*7J
RETURN
END
```



# FUNCTION CUR

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120

FUNCTION CUR PROVIDES AN ESTIMATE OF THE INITIAL MID-HEIGHT CURV.

FUNCTION CUR(OLI)

COMMON R(201), OI(101), R(301), FI(1200), CM(200), LPHN, NSUM, XI(251), Y(1301), RES

COMMON R, QI(1), OI(1), OSI, FIC, AST, AFS, CMX1, CMX2, PNT1

COMMON NN, NRUN, RAN, R4I, NR4, R5N, R5I, NP5, ASTN, ASTI, NAST, AFSN, AFSI

COMMON NAFS, RESN, RESI, MRES, HN, RI, VRB, RIN, R11, NR1, RMAX, PSX1, STFN

COMMON PSX2, RPL, V, W, T, J, DEFA, LL, PSX3, RIN, L, IALI

COMMON C, INUM, REV, EV, C4, PHC, OIR, JA, IR, IC, ID, IE

COMMON DST, SN, CS, JR, C1, C2, C3, C4, C5, C11, PHC1, C13, C14, FMDM, RA, PC, CD

COMMON TIEN, XIEN, FFI, FPI

COMMON LT, IALLA, OI(1A), OI(1CA), NSUM, F1, RK, F2, F3, CH, FS, 71, 72, 73, 74

RK=B/75.-0.30\*RES\*R(111)

IF(R(11)-RK)A3,63,64

63 F1=1.

GO TO 65

64 F1=1.0-((R(11)-RK)/10.0175\*B-0.5731)

IF(F1)60,60,65

69 F1=0.1

65 IF(R(11)-1.166,66,67

66 F2=1.+7.0\*(AST-0.151)\*(1.-R(11))

F3=R(11)

GO TO 68

67 F2=1.0

F3=1.0

68 CUR=0.00004\*B\*B\*F1\*F2\*F3\*OLI-160.1/400.1\*(1.04-1.3\*AST\*RES\*OLI)

1.00107\*OI(1)+10.0075\*R(41)+0.651/R(111)

RETURN

END



# SUBROUTINE PARAM

PAGE  
A-1

```
SUBROUTINE PARAMHS(RBA,RB)
COMMON RI(20),O(10),P(30),FI(200),CH(200),LPUN,NSUM,X(25),Y(30),RFS
COMMON R,DI,PLIC,DST,FIC,AST,AFS,CMY1,CMX2,PNT1
COMMON NN,NRUN,RAN,R4I,NR4,R5N,R5I,MP5,ASTN,ASTI,NAST,AFEN,AFS1
COMMON NAFS,RESN,RESI,NRES,RN,PI,NRB,R1N,R1I,NR1,RMAX,PSX1,STFN
COMMON PSX2,BPI,V,W,T,J,DEFA,LI,PSX3,RIN,L,IALL
COMMON C,INUN,REY,EY,CP,PHC,OIR,JA,IR,IC,ID,IE
COMMON DST,SU,CS,JR,C1,C2,C3,C4,C5,C11,PHC1,C13,C14,FHOM,CA,CC,CD
COMMON TIEM,XIEN,FFIX,FPIN
COMMON LT,IALLA,OL1A,DIICA,NSUMA,F1,RK,F2,F3,GH,FS,71,7J,7H,7L,7M
GH=(-0.0015+R+R+.125+R-5.)*[1.7*AST+.5]
IF(GH-.65)1,3,3
3 HS=0.02*GH
RBA=1.5+R(R1*GH
RB=R(R1*GH
GO TO 2
1 HS=0.013
RB=.65+R(R1
RBA=R(R1
2 RETURN
END
```

# FUNCTION VAL

PAGE  
420

FUNCTION VAL CALCULATES A VALUE USING THE EQUATION OF A PARABOLA  
VAL=((VAL3-VAL2)CA+(VAL1-VAL2)CR1(D)ID)/((CA)(CR1)CA+CR1)+(VAL3-VAL2)(CA)(CA)-(VAL1-VAL2)(CR1)(CR1)ID)/((CA)(CR1)CA+CR1)+VAL2

FUNCTION VAL (VAL1, VAL2, VAL3, VAL4, CA, CR1  
C14=(VAL3-VAL2)\*CA  
C13=CA\*(CR1+CA+CR1)  
C12=(VAL1-VAL2)\*CR1  
C11=(C14-C12)/C13  
C12=(C11\*CA-C12\*CR1)/C13  
VAL=C11\*VAL4+VAL4+C12\*VAL4+VAL2  
RETURN  
END

# FUNCTION BC

PAGE

A22

BY USING NEWTON-S METHOD, FUNCTION BC FINDS THE ZERO ROOT OF A  
PARABOLIC EQUATION SIMILAR TO THAT USED IN FUNCTION VAI

```
FUNCTION BC(RC1,RC2,RC3,RC4)
C3=RC3
C4=RC4
KK=15
0100 C3=RC1+C1+C4+RC2+C4+RC3
C4=C4-C3/12.*RC1+C4+RC21
KK=KK-1
IF(KK)A191,A190,A180
0100 IF(ABS(C3)-0.0000001)A190,A190,A140
0100 PC=C4
RETURN
END
```

# EXAMPLE RUN

RUN NUMBER 77

AST=0.340000 AFS=0.600000 R/T=110.000 AF=0.152239 AW=0.101493

MAX LOAD= 3.190880 STIFF YIELD= 5.314463 PLATE YIELD= 5.314463

PSX1= 2.679400

CMX1= 1.468456

PNT1= 0.738898

AXIAL LOAD= 1.300

PLATE RESID. STRESS=0.075

E/R=0.070

MOD. OF ELASTICITY=29600.0

PLATE YIELD STRESS= 47.000

AP=0.746269 CT=0.202985 SM=0.713682

MAX PL STRS= 2.879400 MAX PL STRN= 5.931998

PSX2=-0.064917

CMY2=-1.141792

*Cont.*

STRAIN INCREMENT= 0.003

STIFF. YIELD STRESS= 47.000

STRAIN	CURVATURE	MOMENT	N	J
1.300	-0.274203	-0.352192	100	3
1.201	-0.332786	-0.309280	99	3
1.102	-0.391853	-0.446088	98	3
1.003	-0.451423	-0.405344	97	3
0.907	-0.500687	-0.542875	96	3
0.811	-0.568459	-0.591058	95	3
0.715	-0.627751	-0.639910	94	3
0.619	-0.687570	-0.689439	93	3
0.553	-0.747000	-0.747000	92	1
0.493	-0.807000	-0.807000	91	1
0.433	-0.867000	-0.867000	90	1
0.373	-0.927000	-0.927000	89	1
0.313	-0.987000	-0.987000	88	1
0.268	-1.055871	-1.029216	87	2
0.250	-1.124199	-1.041418	86	2
0.235	-1.187428	-1.051021	85	2
0.220	-1.257335	-1.060115	84	2
0.208	-1.318810	-1.067026	83	2
0.196	-1.385956	-1.073617	82	2
0.184	-1.459591	-1.079889	81	2
0.175	-1.510662	-1.084385	80	2

*CONT.*

0.166	-1.551125	-1.106451	77	2
0.157	-1.651132	-1.106451	76	2
0.148	-1.731409	-1.106451	75	2
0.139	-1.811708	-1.106451	74	2
0.133	-1.872388	-1.106451	73	2
0.127	-1.933392	-1.106451	72	2
0.121	-2.002476	-1.106451	71	2
0.115	-2.074048	-1.106451	70	2
0.109	-2.150547	-1.106451	69	2
0.103	-2.232528	-1.106451	68	2
0.097	-2.320591	-1.106451	67	2
0.091	-2.415437	-1.106451	66	2
0.085	-2.517883	-1.106451	65	2
0.079	-2.628877	-1.106451	64	2
0.073	-2.749536	-1.106451	63	2
0.070	-2.813892	-1.106451	62	2
0.067	-2.881178	-1.106451	61	2
0.064	-2.951597	-1.106451	60	2
0.061	-3.025373	-1.106451	59	2
0.058	-3.102753	-1.106451	58	2
0.055	-3.184007	-1.106451	57	2
0.052	-3.269432	-1.106451	56	2
0.049	-3.359360	-1.106451	55	2
0.046	-3.454153	-1.106451	54	2
0.043	-3.554220	-1.106451	53	2
0.040	-4.554220	-1.106451	52	2
0.037	-5.554220	-1.106451	51	2
0.034	-6.554220	-1.106451	50	2
0.031	-7.554220	-1.106451	49	2
0.028	-8.554220	-1.106451	48	2
0.025	-9.554220	-1.106451	47	2
0.022	-10.554220	-1.106451	46	2
0.019	-11.554220	-1.106451	45	2
0.016	-12.554220	-1.106451	44	2
0.013	-13.554220	-1.106451	43	2
0.010	-14.554220	-1.106451	42	2
0.007	-15.554220	-1.106451	41	2
0.004	-16.554220	-1.106451	40	2
0.001	-17.554220	-1.106451	39	2
-0.002	-18.554220	-1.106451	38	2
-0.005	-19.554220	-1.106451	37	2
-0.008	-20.554220	-1.106451	36	2
-0.011	-21.554220	-1.106451	35	2
-0.014	-22.554220	-1.106451	34	2
-0.017	-23.554220	-1.106451	33	2
-0.020	-24.554220	-1.106451	32	2
-0.023	-25.554220	-1.106451	31	2
-0.026	-26.554220	-1.106451	30	2
-0.029	-27.554220	-1.106451	29	2
-0.032	-28.554220	-1.106451	28	2
-0.035	-29.554220	-1.106451	27	2
-0.038	-30.554220	-1.106451	26	2
-0.041	-31.554220	-1.106451	25	2
-0.044	-32.554220	-1.106451	24	2
-0.047	-33.554220	-1.106451	23	2
-0.050	-34.554220	-1.106451	22	2
-0.053	-35.554220	-1.106451		

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-0.056	-37.554220	-1.131144	21	
-0.059	-37.554220	-1.131144	20	2
-0.062	-38.554220	-1.131266	19	2
-0.065	-39.554220	-1.131280	18	2
-0.068	-40.554220	-1.131312	17	2
-0.071	-41.554220	-1.131335	16	2
-0.074	-42.554220	-1.131358	15	2
-0.077	-43.554220	-1.131380	14	2
-0.080	-44.554220	-1.131403	13	2
-0.083	-45.554220	-1.131426	12	2
-0.086	-46.554220	-1.131449	11	2
-0.089	-47.554220	-1.131472	10	2
-0.092	-48.554220	-1.131494	9	2
-0.095	-49.554220	-1.131517	8	2
-0.098	-50.554220	-1.131540	7	2
-0.101	-51.554220	-1.131563	6	2
-0.104	-52.554220	-1.131586	5	2
-0.107	-53.554220	-1.131608	4	2
-0.110	-54.554220	-1.131631	3	2
-0.113	-55.554220	-1.131654	2	2
-0.116	-56.554220	-1.131677	1	2
1.402	-0.214331	-0.304301	101	13
1.507	-0.153316	-0.255734	102	13
1.612	-0.092776	-0.207779	103	13
1.716	-0.032692	-0.160407	104	13
1.821	0.026957	-0.113595	105	13
1.926	0.086188	-0.067318	106	13
2.031	0.145021	-0.021555	107	13
2.135	0.203470	0.023717	108	13
2.249	0.266376	0.072228	109	13
2.362	0.328868	0.120209	110	13
2.476	0.390963	0.167680	111	13
2.590	0.452679	0.214663	112	13
2.703	0.514029	0.261177	113	13
2.817	0.575027	0.307239	114	13
2.930	0.635687	0.352867	115	13
3.044	0.696022	0.398076	116	13
3.157	0.756042	0.442882	117	13
3.271	0.815759	0.487298	118	13
3.384	0.875183	0.531338	119	13
3.498	0.934324	0.575015	120	13
3.611	0.993191	0.618339	121	13
3.725	1.051792	0.661322	122	13
3.838	1.110137	0.703976	123	13
3.960	1.172691	0.749552	124	13
4.083	1.234965	0.794770	125	13
4.205	1.298968	0.839630	126	13
4.327	1.358709	0.884171	127	13
4.449	1.420194	0.928375	128	13
4.572	1.481433	0.972262	129	13
4.694	1.542431	1.015840	130	13
4.816	1.603196	1.059110	131	13
4.938	1.663733	1.102106	132	13
5.061	1.724050	1.144809	133	13
5.183	1.784152	1.187236	134	13
5.305	1.844045	1.229390	135	13
5.427	1.903664	1.271212	136	13
5.567	1.963559	1.308410	137	13
5.803	2.022499	1.317071	138	13
6.039	2.080937	1.325134	139	13
6.283	2.141054	1.332921	140	13



5.528	2.2687	1.3466	142	15
6.772	2.250952	1.343313	143	15
7.017	2.318803	1.350275	144	15
7.261	2.377289	1.365075	145	15
7.514	2.437508	1.370528	146	15
7.767	2.497389	1.375663	147	15
8.021	2.556954	1.380507	148	15
8.274	2.616225	1.385084	149	15
8.527	2.675217	1.389415	150	15
8.780	2.733950	1.393518	151	15
9.034	2.792437	1.397411	152	15
9.287	2.850692	1.401233	153	15
9.549	2.910726	1.404862	154	15
9.811	2.970539	1.408312	155	15
10.073	3.030143	1.411595	156	15
10.335	3.089549	1.414723	157	15
10.597	3.148766	1.417705	158	15
10.859	3.207805	1.420552	159	15
11.120	3.266675	1.423272	160	15
11.382	3.325382	1.425874	161	15
11.644	3.383936	1.428363	162	15
11.906	3.442342	1.430748	163	15
12.168	3.500608	1.433109	164	15
12.439	3.558675	1.435370	165	15
12.710	3.620606	1.437539	166	15
12.980	3.680488	1.439619	167	15
13.251	3.740079	1.441617	168	15
13.522	3.799633	1.443536	169	15
13.792	3.859071	1.445381	170	15
14.063	3.918399	1.447156	171	15
14.334	3.977621	1.448865	172	15
14.604	4.036742	1.450511	173	15
14.875	4.095744	1.451682	174	16
15.128	4.155968	1.452162	175	16
15.347	4.215936	1.452621	176	16
15.565	4.275904	1.453062	177	16
15.783	4.335873	1.453484	178	16
16.002	4.395842	1.453890	179	16
16.220	4.455810	1.454280	180	16
16.438	4.515778	1.454309	181	16
16.447	5.515778	1.454338	182	16
16.456	6.515778	1.454367	183	16
16.464	7.515778	1.454396	184	16
16.473	8.515778	1.454425	185	16
16.482	9.515778	1.454454	186	16
16.491	10.515778	1.454483	187	16
16.499	11.515778	1.454512	188	16
16.508	12.515778	1.454541	189	16
16.517	13.515778	1.454570	190	16
16.525	14.515778	1.454599	191	16
16.534	15.515778	1.454628	192	16
16.543	16.515778	1.454657	193	16
16.552	17.515778	1.454686	194	16
16.560	18.515778	1.454715	195	16
16.569	19.515778	1.454744	196	16
16.578	20.515778	1.454773	197	16
16.587	21.515778	1.454802	198	16
16.595	22.515778	1.454831	199	16
16.604	23.515778	1.454860	200	16
16.613	24.515778			

RUN NUMBER 77

\*INPUT\* DS1=0.120000 FIC=0.200000

CASE NUMBER 77. 1 QLL= 40.0000 QIR= 0.00119 PHC1=0.3519

CURV	L/R	X/R	Y/R	MM
0.979072	124.236841	124.229472	0.095676	-1.124545
2.206606	39.576243	89.551195	0.060617	0.

CASE NUMBER 77. 2 QLL= 180.0000 QIR= 0.36537 PHC1=0.5253

CURV	L/R	X/R	Y/R	MM
2.775581	74.022600	74.010978	0.608648	-1.142526
3.344507	54.069902	54.061828	0.430176	0.

CASE NUMBER 77. 3 QLL= 320.0000 QIR= 0.64954 PHC1=0.6970

CURV	L/R	X/R	Y/R	MM
3.400072	57.711360	57.704565	0.414139	-1.141404
4.046881	42.600399	42.595524	0.298548	0.

CASE NUMBER 77. 6 QLL= 110.0000 QIR= 0.22328 PHC1=0.4388

CURV	L/R	X/R	Y/R	MM
2.286803	90.381965	90.364433	0.825809	-1.139711
2.783923	65.139026	65.127152	0.569009	0.

EXIT CALLED.  
CARDS REMAINING IN DECK ARE--  
END OF DATA

AUG 15 67 21 11.3  
RUN TIME 0008.2 MIN.  
AUTHORIZED TIME LEFT IS 013 HR 38.9 MIN



DSII=0.2456 FDI=0.2868

SLOPE	P	FNDS
0.	1.300302	FIXED
0.034591	1.300486	S.S.

DSII=0.1476 FDI=0.2868

SLOPE	P	FNDS
0.	1.301386	FIXED
0.024553	1.300980	S.S.

DSII=0.1119 FDI=0.2868

SLOPE	P	FNDS
0.	1.301677	FIXED
0.021735	1.301209	S.S.

DSII=0.1898 FDI=0.2868

SLOPE	P	FNDS
0.	1.301150	FIXED
0.027506	1.300702	S.S.